Design and Implementation of Two-Degree-of-Freedom Nonlinear PID Controller for a Nonlinear Process

Rakesh M K^1 Satheesh babu. R^2 I. Thirunavukkarasu^{3*}

Abstract: A Two-Degree-of-Freedom (2-DOF) Nonlinear PID (NPID) controller is designed for a conical tank level process. The process is modeled using black box identification method as a First Order Process with Dead Time (FOPDT) model. Two nonlinear first order tracking differentiators are designed for the nonlinear PID controller. The closed loop control is implemented for a wide operating range of the process with the designed controller. The implementation and simulation is carried out using Mat-lab Simulink software. The online implementation of proposed controller shows that 2-DOF NPID is better than 1-DOF NPID.

Index Terms: Two-Degree-of-Freedom, First order process with dead time, Nonlinear PID, Nonlinear tracking differentiator

I. Introduction

A conical tank level process is a nonlinear process due to the conical shape of the tank. The change in area of the conical tank for equal rise in level is nonlinear. The nonlinear process is modeled as piecewise linear models at some operating points. This inherent nonlinear behavior of the processes makes control of such processes more complex. Presence of the nonlinearity in the plant often causes degraded performance of the overall process.

A nonlinear PID controller is developed by defining a nonlinear function which gives the controller settings depending upon the calculated error and some tuning parameters [1][2]. A nonlinear function may contain number of parameters and values of those should be decided by the designer and it varies from plant to plant.

When the process and the controller both exhibit nonlinear characters, a 2-DOF control scheme is considered to improve servo and regulatory problems [3]. In the 2-DOF control scheme the closed loop transfer functions for servo and regulatory can be adjusted independently [4][5][6]. In this work, the tuning of 2-DOF nonlinear PID controller is demonstrated to achieve stability and desired performance measures. This paper is organized such that first the design of the 2-DOF nonlinear PID control structure using the nonlinear tracking differentiators and secondly conical tank level process is described. Then the simulation study and online implementation of designed controller with actual process are shown.

II. 2-DoF Nonlinear PID Control System

A nonlinear function $f(e, \alpha, \delta)$ is used for the nonlinear PID as in [1] given by (1).

1. Nonlinear PID

$$f(e,\alpha,\delta) = \begin{cases} sign(e) | e |^{\alpha}, & when | e | > \delta \\ \delta^{\alpha-1}e, & when | e | \le \delta \end{cases}$$
(1)

The above is a highly nonlinear function with parameters α and δ . The constant δ determines the linear range of the nonlinear function and α decides the shape the nonlinear function. The *sign function* is given in (2).

$$sign(A, \delta) = \begin{cases} sign(A), & when |A| > 0 \\ \frac{A}{\delta}, & when |A| \le \delta, \delta > 0 \end{cases}$$

$$(2)$$

Then, the control law can be written in parallel PID control form as in (3),

$$u = K_p e f_p(e_p, \alpha_p, \delta_p) + K_I f_I(e_I, \alpha_I, \delta_I) \int e dt + K_D f_D(e_D, \alpha_D, \delta_D) \frac{de}{dt}$$
(3)

The NPID controller settings are tuned independently by individual tracking differentiators which makes it 2-DOF control system. The 2-DOF NPID control structure is implemented as shown in Fig.1.

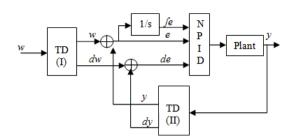


Fig.1. 2-DOF PID Control Structure

The proportional term is selected ($\delta_P \le 1$ and $0 < \alpha_P < 1$) as it should deliver large gain for small error and vice versa. For the integral action in order to avoid the integral saturation $1 \le \alpha_I < 0$ is used. δ_I is the parameter which decides the integral step which can be selected as small as $\delta_I \le 1$. The differential error is proportional to the proportional gain $\alpha_D \le 1$ and δ_D can be selected appropriately [2].

2. Nonlinear Tracking differentiator (NTD)

A tracking differentiator gives the estimate of the derivative of the input and the tracked input. Use of a tracking differentiator makes the differential action more rapid and arrest the influence of noise signal up to some extent [8]. Two first order NTDs are used in the control structure, one for the set-point and the other in feedback path for the process variable.

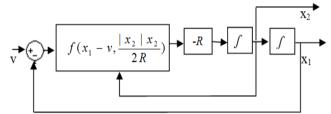


Fig.2. First order nonlinear tracking differentiator

Schematic representation of a first order NTD is given in Fig.2. A nonlinear function of input and two outputs by means of a feedback is combined to building NTD block and the same mathematically represented as follows in (4), where R is the tuning parameter.

$$\begin{cases} \frac{dx_1}{dt} = x_2\\ x_2 = -R \ sign(x_1 - v(t) + \frac{|x_2| x_2}{2R})\\ & \text{III.} \quad \text{System Description} \end{cases}$$



Fig.3. Conical Tank Level Process

(4)

A Conical tank level process setup is shown in Fig.3. The conical tank is of 30cm height. The final control element is a DC motor pump which can deliver max inflow of 1600 lph. A differential pressure transmitter is used to measure the level. The obtained process model is shown below in (5) which is modeled as F

$$P(s) = \frac{.925}{25.05s + 1}e^{-1.09s} \tag{5}$$

A conical tank level process is a nonlinear process due to the conical shape of the tank. The change in area of the conical tank for equal rise in level is nonlinear [9]. This inherent nonlinear behavior of the processes makes control of such processes more complex. Presence of the nonlinearity in the plant often causes degraded performance of the overall process and the process are modeled as piecewise linear models at some operating level points.

IV. Implementation and Results

The tuned NPID parameter values are given in table I. The proportional gain (K_P) with lesser values makes the process variable overshoot and higher value of the same cause steady state error. The large values of integral gain (K_I) make the system asymptotically stable. K_P and K_I values are chosen suitably as 15 and 1.9. The derivative gain is chosen preferably very low to avoid the chattering effect. The parameters α_P , α_I , α_D , δ_P , δ_I , δ_D , δ_1 , and δ are defined in the range of zero and one.

Table I. Controller settings								
1-DOF	⁷ NPID	2-DOF NPID						
K _P	15	K _P	15					
KI	1.9	KI	1.9					
K _D	.01	KD	.01					
$\alpha_{\rm P}$	0.5	$\alpha_{\rm P}$	0.5					
$\alpha_{\rm I}$	0.5	$\alpha_{\rm I}$	0.5					
$\alpha_{\rm D}$	0.5	$\alpha_{\rm D}$	0.5					
δ_P	1	δ_P	1					
δ_{I}	1	δ_{I}	1					
$\delta_{\rm D}$	1	$\delta_{\rm D}$	1					
-	-	R ₁	10					
-	-	R ₂	30					
-	-	δ_1	.01					
-	-	δ_2	.1					
-	-	δ_2	.1					

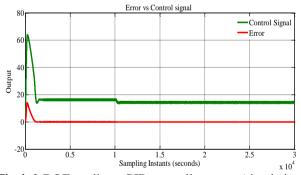


Fig.4. 2-DOF nonlinear PID controller output(simulation)

A nonlinear function may contain number of parameters and value of those should be decided by the designer and it varies from plant to plant. The simulated closed loop responses for a nominal value of 15cm are shown in Fig. 4 and 5 respectively. A smooth nonlinear control action for the nonlinear level process with the 2-DOF NPID makes the process variable to approach the set point immediately. 15% of nominal value is given as positive load and the loas is applied at 100s.

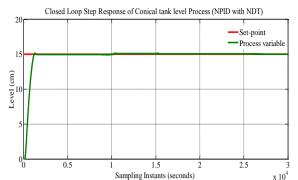


Fig.5. Simulated closed loop step response of process model with 2-DOF NPID Controller

The controller settings are tuned in simulation by reducing the integral absoulte error and the parameters are obtained as shown in table I. The nonlinear PID controller designed without an explicit model is implemented to control the whole operating range (0-30 cm) of conical tank.

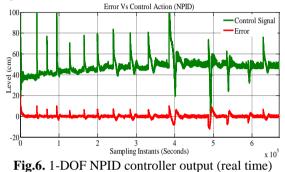


Fig. 6 and 7 show that the control effort taken by the 1-DOF and 2-DOF nonlinear PID control schemes implemented on closed control of conical tank level process. The use of nonlinear NTD in 2-DOF control scheme reduces the effect of noise and chattering in system steady state response.

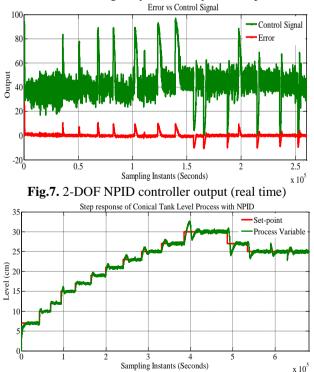


Fig.8. Closed loop response of conical tank level process with 1-DOF NPID controller (real time)

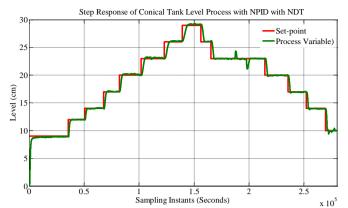


Fig.9. Closed loop response of conical tank level process with 2-DOF NPID controller (real time)

The real time plant response shown in Fig.8 with 1-DOF nonlinear PID controller corresponds to the control action in Fig.6 is. The response is having large overshoots especially at high operating points may end up in wind up effect while large set point are applied.

1-DOF NPID				2-DOF NPID						
Level (cm)	Tr	Ts	%M	Level (cm)	Tr	Ts	%			
	(s)	(s)	Р		(s)	(s)	MP			
0-7	17	206	5	0-9	11	70	0			
7-10	10	170	20	9-12	12	58	0			
10-12	8	275	30	12-14	20	121	7			
12-15	18	172	40	14-17	11	80	5			
15-17	22	189	25	17-20	12	86	9			
17-19	21	134	40	20-23	25	144	0			
19-21	26	224	60	23-26	32	79	10			
21-23	28	344	65	26-29	42	138	0			
23-25	30	155	65	29-26	29	43	0			
25-27	32	336	67	26-23	40	72	0			
27-30	46	689	95	23-20	38	70	0			
30-27	44	379	102	20-17	28	57	0			
27-25	38	362	98	17-14	24	42	0			
% Load	Ts			% Load	Ts					
applied	(s)			applied	(s)					
+10	190			+10	29					
-10	207			-10	43					

Table II. Controllers performances

The closed loop step response of the real time process with the 2-DOF nonlinear PID is shown in Fig.9. The plant tracks the set point efficiently with a satsfactory settling time. The over shoot and under shoot is arrested by the 2-DOF nonlinear control scheme. Both positive and negative load is given when the process is settled at 23cm. The load rejection is done effectively by the nonlinear PID as it can be observed in Fig.6. Camparison of designed controllers performances are provided in table II. Time domain performances are shown as T_r -rise time, T_s -settling time, MP-Overshoot. Load is applied as percentage of operating level.

V. Conclusion

The performance measures of 1-DOF and 2-DOF nonlinear PID controllers which are designed for the described conical tank level process are calculated and compared. It is found that the 2-DOF nonlinear control scheme is superior over 1-DOF by arresting the overshoot significantly and reducing the settling time and rise time. The modeling errors also can cause some complexity in control of processes especially nonlinear in nature The designed 2-DOF nonlinear PID control scheme is found as robust with the real time plant by offering excellent servo and regulatory responses. Apart from the difficulty in tuning of the control parameters the control scheme is found as improving the performance of 1-DOF nonlinear and conventional control schemes.

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- ^{1.} PG Student, Department of I&C, MIT, Manipal, India. Email: rakeshmk@outlook.com
- ². JRF, Department of I&C, MIT, Manipal, India.
- ^{3*} Associate Professor, Department of I&C, MIT, Manipal, India.