

## **Study on Natural Convection Heat Transfer In an Enclosure— A Review**

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**Abstract:** *Natural convection heat transfer in an enclosure is used in several technological applications for the dissipation of generated heat from the equipment. There are so many researches are conducting on the above same topic in different aspects by changing the position of the heated plate and the boundary conditions of the enclosure. This paper deals with the study of natural convection heat transfer in an enclosure with a heated plate at different position and different boundary conditions. The correlations generated by the researches are presented.*

**Keywords:** *Natural Convection, Heat Transfer, Heat Transfer Correlations*

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### **I. Introduction**

Natural convection is considered as a preferable heat transfer method wherever possible due to its easiness, loyalty, and cost effectiveness. Thermal management of electronics equipment is an active area of research in consideration of the fact that heat dissipation in electronic components has gone up many overlap with every new design. The heat induced by electronic components must be blown to prevent urgent failure and improve long term authenticity. The increasing heat generation rate also add to potential reliability concerns, as failure of these systems consistently results in device failures. The cooling by natural convection is considered as the most effective and efficient cooling mechanism. Application of active cooling techniques is limited because of the additional design resulting in higher operating costs.

### **II. Literature Survey**

M.R.Rajkumar, G.Venugopal and S. Anil Lal[1] experimentally and numerically studied about the natural convection from free standing tandem planar heat sources in a vertical channel. Result showed that Effect of side confining walls is found inconsequential on heat transfer for aspect ratio  $< 0.21$ . Rajesh Baby and C. Balaji[2] experimentally studied about the investigations on phase change material based finned heat sinks for electronic equipment cooling. They found that the effectiveness of a PCM based heat sink depends on the volumetric percentage of the TCE, fin configuration, TCE material, the amount of PCM, heat source power level and so on.

Claudio Cianfrini et al[3] studied Natural convection in tilted square cavities with differentially heated opposite walls and concluded that, for a sufficiently wide range of  $\gamma$  around  $135^\circ$  the overall amount of heat transferred in the  $x$ -direction across the cavity is larger than that corresponding to the untilted case. Mostafa Mahmoodi[4] did Numerical simulation of free convection of nanofluid in a square cavity with an inside heater and found that, for the vertical heaters, the rate of heat transfer is directly proportional to the length of heater for all range of Rayleigh numbers considered.

S.A. Nada[5] conducted an experiment on Natural convection heat transfer in horizontal and vertical closed narrow enclosures with heated rectangular finned base plate and results showed that introduction of fins with any fin array geometries increases the rate of transfer of heat. M.J. Sable et al[6] investigated an enhancement of natural convection heat transfer on vertical heated plate by multiple v-fin array and new heat transfer technique is found out to increase the effect of natural convection heat transfer on a heated plate which is vertical. Lahcen Boukhattem [7] conducts a numerical study on heat transfer in a room with internal heated plate. He conducts this experiment by keeping two vertical walls insulated and two horizontal plate at a constant temperature. And he found that heat transfer from the heated plate is mainly directed to the top wall.

Manmatha K Roul [8] conducted an experiment on natural convection heat transfer through a vertical tube. In his work he conducts his experiment on an aluminum tube. And cover the tube with insulating materials and after observe the heat transfer rate he found that the length of the channel ring cheekiness and ring spacing has a significant effect on heat transfer. Hisanobu Kawashima [9] he conducts an experimental and numerical study on heat transfer on a cooling plate with and without grooves on the plate. He conclude that the grooved

plate reduces the heat transfer due to the formation of vortex inside the grooves. S. Anil Lal [10] conducts a numerical study on natural flow of air in a room. He constructs a 2-D room with two sets of sloped roof at two levels and four openings on both sides and assumes the east side roof receives solar radiation heat and a part of it is transmitted into the room and he found that the speed of air entering the room increases with Rayleigh number and leaves through the top opening at the same rate and the heating of air occurs only in a small region near the heated wall.

M. Al-Arabi [11] conducted an experimental and numerical study on natural convection heat transfer from an isothermal horizontal plate for different shapes and he arrived at expressions for Nusselt number in terms of Rayleigh number, and also he found that the average heat transfer coefficient from a circular plate is almost equal to the heat transfer coefficient for the square plate, only when the heat transfer at the corner is negligible.

Stuart W. Churchill [12] makes a correlation for laminar and turbulent type convective heat transfer from a vertical plate. He derived the expressions for Nusselt number in terms of Rayleigh number and Prandtl number for inclined vertical plates up to  $60^\circ$ . R. Ruiz [13] conducted an experimental study on natural convection heat transfer in V-shaped and L-shaped corners by using water as the heat transfer medium. He found that the value of Nusselt number is greater for V corner than single inclined plate and also Nusselt number is greater for horizontal wall of L corner than single horizontal plate. And when comparing L and V shapes L corner has higher value for Nusselt number. P. Kandaswamy [14] conducted an experimental and numerical study on natural convection heat transfer in a square cavity in the presence of a heated plate placed in vertical and horizontal direction and also for different aspect ratios. He concludes that as the aspect ratio increases the heat transfer also increases and the heat transfer rate for vertical plate is higher than horizontally placed heated plate.

Mostafa Mahmoodi [15] conducted a numerical study on natural convection heat transfer of various water-based Nano fluids in a square cavity with a thin heated plate. He found that for high values of Rayleigh number the water-based Nano fluid is more effective in heat transfer and for low values of Rayleigh number the water-based Nano fluid is ineffective. Zakeriya Altac [16] conducted an experimental and numerical study on natural convection heat transfer from a thin horizontal isothermal plate in an air-filled cavity. He found that maximum heat transfer is possible when the plate is placed near the bottom of the enclosure and also he made a correlation for Nusselt number in terms of Rayleigh number.

Sir W. Wu, D. Ewing & C. Y. Ching [17] conducted a study on the topic "The effect of top and bottom wall temperature on the laminar natural convection in an air-filled square cavity. In this study the experiment is done by using a square cavity of 305 mm height and width of 305 mm and the depth of the cavity is 914 mm. One vertical and one top wall are heated and other vertical and bottom wall are cooled. The heated walls are constructed by using an aluminium plate. The temperature of the heated and cooled wall are noted by using a T-type thermocouple. From this study he concludes that the changes in the top and bottom wall temperature also affect the temperature of the air outside the thermal boundary layer in the square cavity.

Tanmy Basak, S. Roy, A. R. Balakrishnan [18] conducted a study on the topic "The effect of thermal boundary condition on natural convection flow within a square cavity." This experiment is conducted to study the laminar natural convective flow in a square cavity. The bottom wall is uniformly and non-uniformly heated and the top wall is maintained at constant temperature. The experiment is a prototype of different industrial applications such as operation and safety of nuclear reactor and convective heat transfer associated with boiler. From this study he concludes that the thermal boundary layer developed over 80% of the cavity in uniform heating and in non-uniform heating the thermal boundary layer developed is only 60%. Zeev Rotem and Lutz Claassen [19] conducted a study on the topic "Natural convection above unconfined horizontal surface." This paper deals with the free convective flow above a horizontal plate. The experiment is conducted to demonstrate the existence of laminar boundary layer above a horizontal plate at intermediate Grashof number.

A. Daloglu and T. Ayhan [20] conducted a study on the topic "Natural convection in a periodically finned vertical channel." This paper presents the experimental results of natural convection in a rectangular cross-sectional vertical channel. The fins are connected to the channel and both plates are placed periodically. The uniform heat flux is maintained on the channel. He concludes that natural convection in a finned rectangular cross-section vertical channel is investigated and results compared with smooth channel experimentally. I. Dagtekin and H. F. Oztop [21] conducted a study on the topic "Natural convection heat transfer by heated partitions within an enclosure." In this study natural convection heat transfer and fluid flow of heated partition within an enclosure have been studied numerically. The right side wall and bottom wall are insulated and left side wall and top side wall are maintained at same temperature.

Witold M. Lewandowski, Ewa Radziemska, Buzuk [22] conducted a study on the topic "Free convection heat transfer and fluid flow above horizontal rectangular plate." This topic is based on the study of natural convective heat transfer from horizontal rectangular isothermal plate having different aspect ratios. A three-dimensional physical model was proposed for convective fluid flow above heated rectangular plate and analytical expressions for the dependence of the Nusselt number and the Rayleigh number were obtained as

solution from the model M.R Rajkumar ,G.Venugopal,S.AnilLal [23]conduct a study on the topic natural convection with surface radiation from a planar heat generating element mounted freely in a vertical channel . in this study experimental and numerical stimulation is done and heat transfer by natural convection and radiation from a planar heat generating element placed centrally between two adiabatic vertical plate are studied C.E kwak and T.H Song[24]conduct a study on the topic experimental and numerical study on natural convection from vertical plate with horizontal rectangular grooves. In this study natural convection from two dimensional vertical plate with horizontal rectangular groove was studied experimentally and numerically

Giovanni Tanda[25] conducts experiment on natural convection from two staggered vertical plates and found that ,the input power provided to each plates is changed to natural convection and radiation Kenzo Kitamura, Fumiyoshi Kimura[26] studied the Heat transfer and fluid flow of natural convection adjacent to upward facing horizontal plates and found that On the leading edges of the plate heat transfer coefficients are largest and reduces with distance. The turbulent fluid motion was created irregular and unsteady temperature. On the central region of the plate. C.Balaji and S.P Venkateshan[27] studied Interaction of radiation with free convection in an open cavity and a Correlation created for convective and radiative transfer based on numerical results. M.Fuji et al[28] studied on Natural convection to air from an array of vertical parallel plates with discrete and protruding heat sources and finally found that Numerical results satisfy with experimental results except where three dimensional effects occur. L.Boukhattem et al[29] carried out a numerical simulation of thermal convection in a closed cavity in the presence of thin horizontal heated plate and results obtained is that the heat transfer from heated plate to isothermal horizontal wall is flows to top wall

M.R Rajkumar[30] studied on experimental correlation for multimode heat transfer developed using asymptotic expansion method and found that, when emissivity increases temperature of heat source decreases.

Anu Nair et al [31] conducted an review of Natural Convective Heat Transfer From Horizontal Heated Plate Facing Upward In Vertical Channel. He compared correlation for Nusselt number in terms of Rayleigh number with different working fluid in different shapes of plates in detailed manner. Anu Nair et al. [32,33] Conducted new Optimization method (i. e, Least square method and Bayesian Approach ) to find out the Value of C in the Nusslet Number.. Anu Nair et al [34,35] conducted a numerical investigation of natural convection flow and heat transfer in a square enclosure due to heat source or sources on its left wall has been investigated. M.Ebenezer et al[36]studied Determination of hot spot temperature of a single phase transformer subjected to harmonic pollution and conclude that, When number of cells increases average surface temperature increases. From numerical analysis temperature distribution and location of hot spot temperature is determined.

### III. Comparision of Correlations

From the literature survey of natural convection heat transfer from a heated plate of different geometry we got some correlation. They are listed below.

SI no	Researchers	Investigation	Fluid	Plate geometry	correlations	Buoyancy strength
1	M. Al-Arabi and M. K. El-Riedy [11]	Experiment	Air	Rectangular (L/W =1 to 4)	$Nu = 0.700Ra^{1/4}$ $Nu = 0.155Ra^{1/3}$	$2 \times 10^5 \leq Ra \leq 4 \times 10^6$ $4 \times 10^7 \leq Ra \leq 10^8$
2	Stuart W. Churchill and Humbert S. Chu [12]	Experiment	Air	Inclined rectangular	$\bar{Nu}^{\frac{1}{2}} = \left\{ 0.825 + \frac{0.38Ra^{\frac{1}{4}}}{\left[ 1 + (0.49/Pr)^{\frac{9}{16}} \right]^{\frac{8}{27}}} \right\}^2$	$0 \leq \theta \leq 60^\circ$
3	Zekeniya Atlac and Seda Konrat [16]	Analytic	Air	Rectangular (L/W = 0.5-.075)	$Nu = 0.2354Ra^{0.2085} A^{0.752} Y_0^{-0.222} \left( \frac{L}{l} \right)^{0.343}$	$10^5 \leq Ra \leq 10^7$ $0.25 \leq Y_0 \leq 0.7$ $0.5 \leq l/L \leq 0.7$
4	C.Balaji and S.P Venkateshan [27]	Experiment	Air	Vertical rectangular	$Nu = 0.425 \left( Gr_H^{0.254} \left( \frac{N_{RC}}{N_{RG1}} \right)^{181} + (1 + \varepsilon^{-0.039}) \right)$	$X = 0.0, Y \geq 0$ $Y = 0.0, X \geq 0, A \leq 1$

## Nomenclature

Nu	Nusselt number
Pr	Prandtl number
Ra	Rayleigh number
Gr	Grashoff number
W	width
h	Coefficient of heat transfer
k	Thermal conductivity
L	Length of the plate
x	Distance from the which plate is placed
$\theta$	Angle of inclination

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