

Parametric Optimization of Variable Compression Ratio Diesel Engine Fueled With Palm Seed Biodiesel and Its Blend Using the Taguchi Method for Mechanical Efficiency & Brake Thermal Efficiency

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Abstract: This study was initiated to find the Mechanical Efficiency & Brake Thermal Efficiency of single cylinder diesel engine operating on palm seed biodiesel and compare them with Mechanical Efficiency and Brake Thermal Efficiency of engine when operated on pure diesel and blend of 50% diesel and 50% palm seed biodiesel. The aim of this study was to determine and compare the effects of variable parameters like Blend ratio, Compression ratio, Injection pressure and load on performance parameter, Specific fuel consumption. Since there are higher number of parameters and levels, the number of probable models are high. So, for selection of optimum parameters, huge analysis work is required which takes so much time. To neglect this problem, Taguchi method is used. The set of combination for factors and levels are achieved with the help of orthogonal array.

Keywords: Palm seed Biodiesel, Compression ratio, Load, Injection pressure, Specific Fuel Consumption, Mechanical Efficiency, Brake thermal Efficiency, Taguchi method, Orthogonal Array

I. Introduction

There are so many environmental and pollution problems today. Fuels prices are also increasing day by day. In this scenario it is needed to find an alternative fuel for internal combustion engines which is producing less pollution and are lower in price. Several alternate fuels are tried by various researches. An experimental study has been carried out for palm seed biodiesel blended with diesel used in a single cylinder diesel engine. Palm seed biodiesel presents a very promising scenario of functioning as alternative fuels to fossil diesel fuel. The properties of these can be compared favourably with the characteristics required for internal combustion engine fuels especially direct injection (diesel) engine. In this work, Taguchi method is used for doing optimization. Design of experiments (DOE) is a systematic method to determine the relationship between factors affecting a process and the output of that process. In other words, it is used to find cause-and-effect relationships. This information is needed to manage process inputs in order to optimize the output. So here, set of combination for varying parameters are obtained by orthogonal array. Orthogonal array is very important in this method. It helps in studying total number of variables by decreasing number of trials. Different Orthogonal Arrays are available in Minitab software which is designed by number of factors and levels. In this work, four factors and 2-3-level design is adopted. So here, we have selected L9 (3⁴) design. In which no. of experiments are 9, no. of factors are 4 and levels are mixed 2-3 level. Then results are obtained by doing experiments as per above method and optimum set of input parameters are found which has minimum Specific Fuel Consumption.

II. Literature Review

Shahbazi et al. (2012) studied that synthesized biodiesel has the properties comparable with the conventional diesel fuel and can be considered as an alternative to mix with it and use in motors [1]. Mofijur et al. (2014) studied that as the reserves of fossil energy resources are becoming limited and decreasing day by day, discovering alternative energy sources has been becoming one of the global agenda. They said that biodiesel has been gaining popularity among the researchers as one of the alternative energy sources due to potential capacity of reducing dependency on fossil diesel fuel, decreasing environmental pollutants, etc. [2]. Tushar Patel et al. (2015) studied that in the statistical design of experiments, Taguchi method is a collection of a mathematical and statistical techniques useful for the parametric optimization and analysis of problem in which a response of interest is influenced by several variables and the objective is to optimize this response [3]. Tushar Patel et al. (2015) studied that Taguchi method is a simplest method of optimizing

experimental parameters in less number of trials. The number of parameters involved in the experiment determines the number of trials required for the experiment [4]. **Mustafa et al. (2016)** investigated on Taguchi's design of experiment method and analysis of variance (ANOVA) applied in order to find optimum operating parameters giving the best engine performance and exhaust emissions with a minimum number of the engine tests in a spark ignition (SI) engine fueled with pure gasoline, ethanol and methanol. They said that Taguchi method makes statistical design by using the orthogonal array and also offers an opportunity to reduce the number of experiments. Taguchi technique computes a signal-to-noise (S/N) ratio based on experimental data [5].

III. Methodology

It is needed to adopt such a methodology that it increases efficiency and decreases specific fuel consumption. Taguchi's design approach employs two designs called the inner and outer array. The Taguchi experiment is the cross product of these two arrays. A Taguchi experiment repeats the outer array design for each run of the inner array. The response variable in the data analysis is not the raw response or quality characteristic; it is the signal-to-noise ratio. Figure 1 shows the flow chart of Taguchi method.

Steps of the experiment:

- 1-Defining goal
- 2-Selecting Parameter
- 3-Selecting level
- 4-Selecting orthogonal array
- 5-Conducting Experiment
- 6-Plotting curves and doing performance analysis

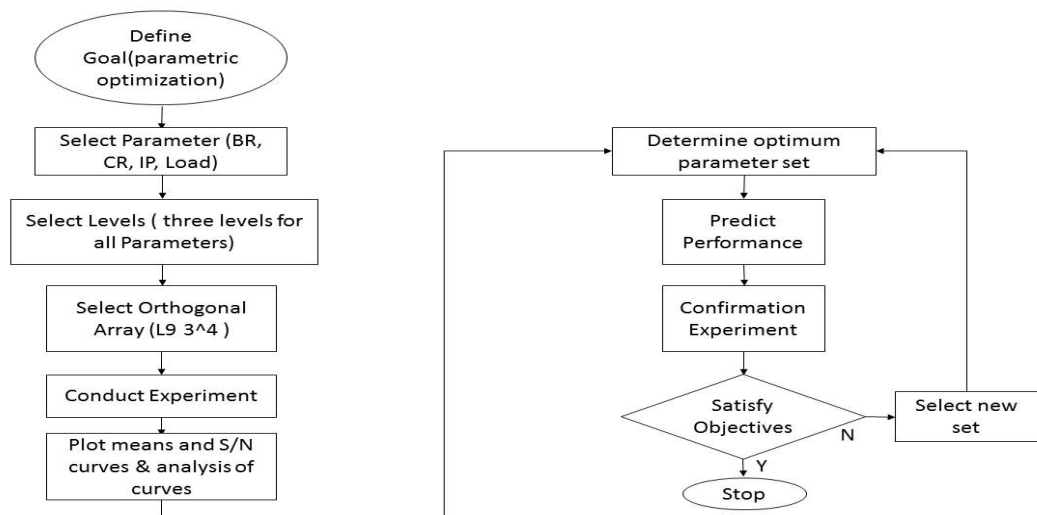


Fig 1: Flow chart of Taguchi method [6]

IV. Factors and levels

In this practical, four factors and 2-3-level design is adopted. So here, we have selected L9 (3⁴) orthogonal array design. In which no. of experiments are 9, no. of factors are 4 and levels are mixed 2-3 level. The variable parameters can be varied within specified range. Here, we have taken Fuel, Blend ratio, Compression ratio, Injection pressure, and Load as variable parameters. The measuring parameters are Brake power, Brake specific fuel consumption, Brake thermal efficiency, Indicated thermal efficiency, Mechanical efficiency, Volumetric efficiency. The combination of factors and levels are as shown in table 1.

Table 1: Factors & levels

Factors	Level 1	Level 2	Level 3
Blend Ratio	D100B0 (Pure Diesel)	D50B50 (50% Blend)	D0B100 (Pure Palm Seed Biodiesel)
Compression Ratio (CR)	14	16	18
Injection Pressure (Bar)	160	175	180
Load (Kg)	1	5	9

V. Experimental set up

The set up consists of single cylinder, four stroke, and multi-fuel research engine in which loading has been provided by eddy current type dynamometer. If change from Petrol to Diesel or Diesel to Petrol is needed, it can be done by doing some necessary changes. In both modes, compression ratio has been varied without changing geometry of combustion chamber and without stopping the engine. For this, special tilting cylinder block arrangement has been designed. There is facility for changing injection point and spark point. Necessary instruments for combustion pressure, Diesel line pressure and crank-angle measurements have been provided in set up. These signals are interfaced with computer for pressure crank-angle diagrams. For interface airflow, fuel flow, temperatures and load measurements, required instruments have been given. The set-up has stand-alone panel box consisting of air box, two fuel tanks for dual fuel test, manometer, fuel measuring unit, transmitters for air and fuel flow measurements, process indicator and hardware interface. For cooling water, and calorimeter water flow measurement, Rotameter has been used. A battery, starter and battery charger have been provided for engine electric start arrangement. The setup enables study of VCR engine performance for brake power, indicated power, frictional power, BMEP, IMEP, brake thermal efficiency, indicated thermal efficiency, Mechanical efficiency, volumetric efficiency, specific fuel consumption, A/F ratio, heat balance and combustion analysis. Lab view based Engine Performance Analysis software package “Enginesoft” has been provided for on line performance evaluation. The testing rig is as shown in figure 2. Table 2 shows the list of sensors and the list of engine specifications are as shown in table 3 [8].

Table 2: List of sensors

F1 to F6	Flow sensor for Fuel and Air
W	Load sensor
N	Engine Speed Sensor
PT	Cylinder Pressure Sensor, Injection Pressure Sensor
T1 to T6	Temperature Sensors

Table 3: Engine setup specifications[7]

Engine manufacturer	Apex Innovations (Research Engine test set up)
Software	Engine soft Engine performance analysis software
Engine type	Single cylinder four stroke multi fuel research engine
No. of cylinder	1
Type of cooling	Water cooled
Rated Power	3.5 kW @ 1500 rpm
Cylinder diameter	87.5 mm
Orifice diameter	20 mm
Stroke length	110 mm
Connecting rod length	234 mm
Dynamometer	Type: eddy current, water cooled, with loading unit



Fig 2: Engine testing rig

VI. Results And Discussion

The results of BSFC are analyzed using Minitab 17. Minitab offers four types of designed experiments: factorial, response surface, mixture, and Taguchi (robust). The steps followed in Minitab to create, analyses, and graph an experimental design are similar for all design types. After conducting the analysis and entering the results, Minitab provides several analytical and graphing tools to understand the results. The S/N ratio for optimal BSFC are coming under “Smaller-is-better” characteristic, which can be calculated as logarithmic transformation of the loss function. Experiment has been done for the four parameter which have been selected and sets of parameters by Minitab software and results have been found for those sets of parameters. Results for those sets are as shown in table 4.

Table 4: Result table for Mechanical & Brake Thermal Efficiency

Ex. No.	Load (kg)	Torque (Nm)	IP (kW)	BP (kW)	FP (kW)	$\eta_{ithe}(\%)$	$\eta_{bthe}(\%)$	$\eta_{mech.}(\%)$
1	1.03	1.87	3.43	0.3	3.13	64.13	5.6	8.73
2	5.14	9.33	4.59	1.48	3.12	59.4	19.09	32.13
3	8.87	16.1	5.21	2.47	2.74	48.65	23.1	47.47
4	8.93	16.21	5.35	2.42	2.93	52.89	23.92	45.22
5	0.89	1.61	3.07	0.25	2.82	57.66	4.75	8.23
6	5	9.07	4.38	1.44	2.94	52.82	17.38	32.9
7	4.82	8.75	4.46	1.38	3.08	58.03	17.95	30.93
8	8.86	16.09	5.35	2.49	2.86	52.75	24.54	46.53
9	0.96	1.75	3.13	0.28	2.84	55.45	4.98	8.99

VII Analysis for Mechanical Efficiency:

Main Effects Plot for S/N ratio data and Mean data for Mechanical Efficiency are shown in Figure 3&4 below.

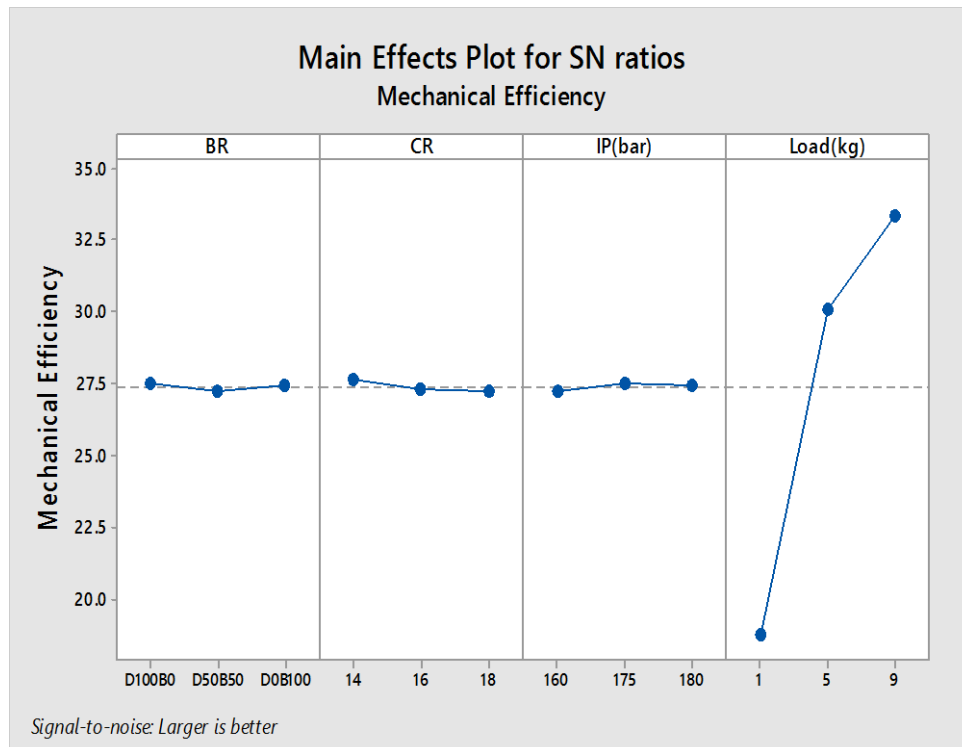


Fig 3: Main Effects Plot for SN ration for Mechanical Efficiency

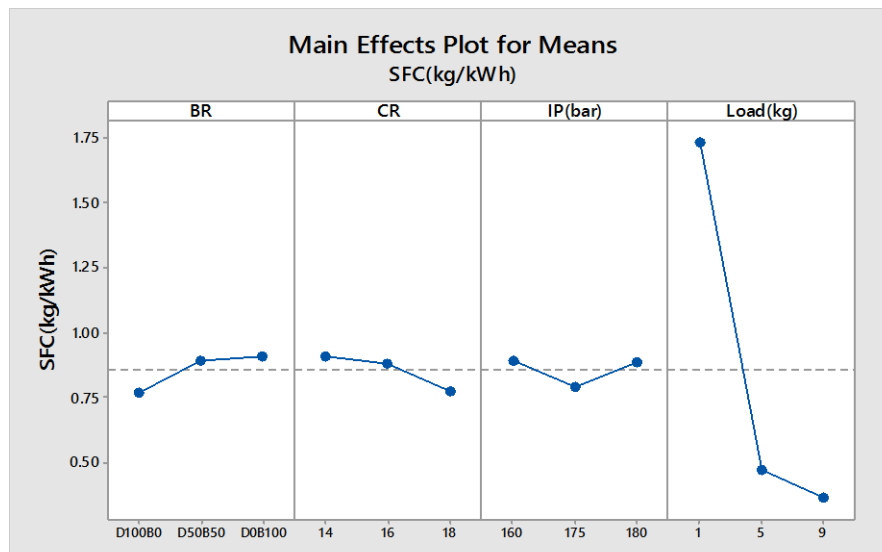


Fig 4: Main Effects Plot for Means for Mechanical Efficiency

Table 5 shows the Response table for Signal to Noise ratio(S/N) for Mechanical Efficiency.

Table 5: Response table for Signal to Noise ratio(S/N) for Mechanical Efficiency

Level	BR	CR	IP(bar)	Load(kg)
1	27.50	27.65	27.21	18.73
2	27.25	27.27	27.51	30.10
3	27.41	27.24	27.44	33.33
Delta	0.24	0.40	0.29	14.60
Rank	4	2	3	1

Referring figure 3, in which we have considered larger is better, we get largest values of SN ratio for blend ratio for pure diesel (D100B0) which is 27.4956. We get same for compression ratio of 14 which is 27.6492. We get same for injection pressure of 175 bar which is 27.5063. And we get maximum SN for load of 9 kg which is 33.3299. Referring figure 4, we get highest value of mean for blend ratio of pure diesel (D100B0) which is 29.4433. We get highest value of mean for compression ratio of 14 which is 29.7867. We get highest value of mean for injection pressure of 175 bar which is 29.3867 and we get highest value of mean in load for load of 9 kg which is 46.4067. And, we get the lowest value of mean for blend ratio of blend of 50% diesel and 50% palm seed biodiesel (D50B50) which is 28.7833. We get lowest value of mean for compression ratio of 18 which is 28.2933. We get lowest value of mean for Injection pressure of 180 bar which is 28.78 and we get lowest value of mean in load for load of 1 kg which is 8.65.

In table 5, level 1 is for the readings of three experiments performed on pure diesel (D100B). BR for level 1 shows average value of blend ratio for these three readings for diesel which is 27.50. CR shows average value of compression ratio for these three readings for diesel which is 27.65. As same, average IP is 27.21 and average load is 18.73. Now values for level 2 and level 3 indicate same things. Delta is the difference between highest and lowest reading from all levels for particular parameter. And, according to these delta values, the ranks are given from highest delta to lowest delta as shown in table. The S/N Ratio for different performance response are calculated at each factor level and the average effect are determined by taking the total of each factor level and divided by the number of data points in the total. The greater the difference between S/N ratios values, the more is parametric influence. The parameter level having the highest S/N ratio corresponds to the sets of parameters which indicates highest performance. Because of, we have considered it for mechanical efficiency which must be as high as possible, we need to consider larger is better factor. So, from above discussion, we obtain table 6 for optimized set.

Table 6: Optimized set of parameter for Mechanical Efficiency

BR	CR	IP(bar)	Load(kg)	$\eta_{mech.}$ (%)	SN ratio
D100B0	14	175	9	47.98	33.8200

From above table, we have been getting optimized set for pure diesel with compression ratio of 14, injection pressure of 175 bar, load of 9 kg and mechanical Efficiency of 47.98 % and we get SN ratio of 33.8200. As experiment has been done on parametric optimization of CI Engine using palm seed biodiesel, it is necessary to find the performance parameters by using palm seed biodiesel for above optimized set. In table 7, optimized set for palm seed biodiesel is shown.

Table 7: Optimized set of parameter for Mechanical Efficiency for Palm Seed Biodiesel

BR	CR	IP(bar)	Load(kg)	$\eta_{mech.}$ (%)	SN ratio
D0B100	14	175	9	47.3533	33.7369

From above table, we have been getting mechanical efficiency of 47.3533% and SN ratio of 33.7369 for optimized set for palm seed biodiesel with compression ratio of 14, injection pressure of 175 bar, load of 9 kg. Table 8 shows validation results for mechanical efficiency.

Table 8: Validation Results for Mechanical Efficiency.

Validation Results for Specific fuel Consumption	
For pure Diesel	
Predicted Value	Experimental Value
47.98	46.30
For Palm Seed Biodiesel	
Predicted Value	Experimental Value
47.3533	46.70

VI.II Analysis for Brake Thermal Efficiency:

Main Effects Plot for S/N ratio data Mean data and for Brake Thermal Efficiency are shown in Figure 5&6 below.

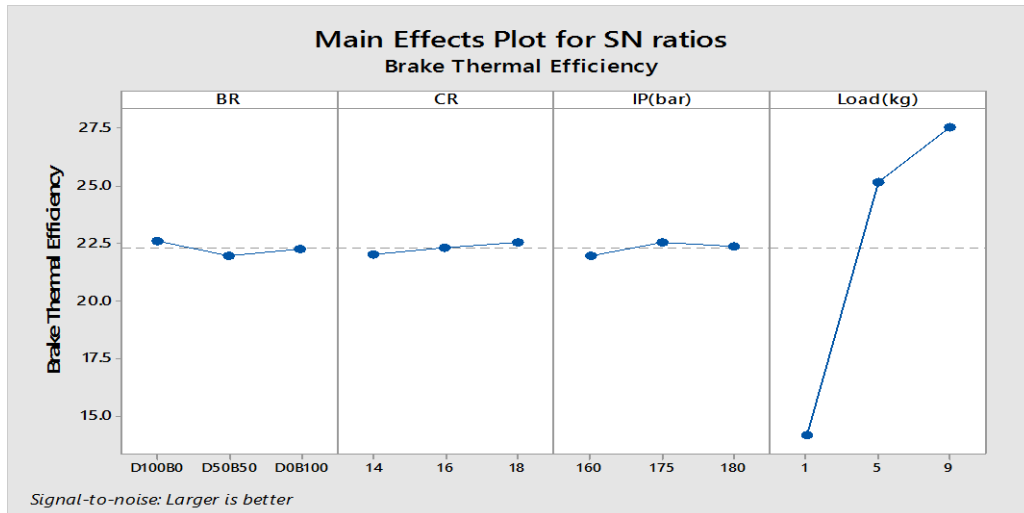


Fig 5: Main Effects Plot for SN ratio for Brake Thermal Efficiency

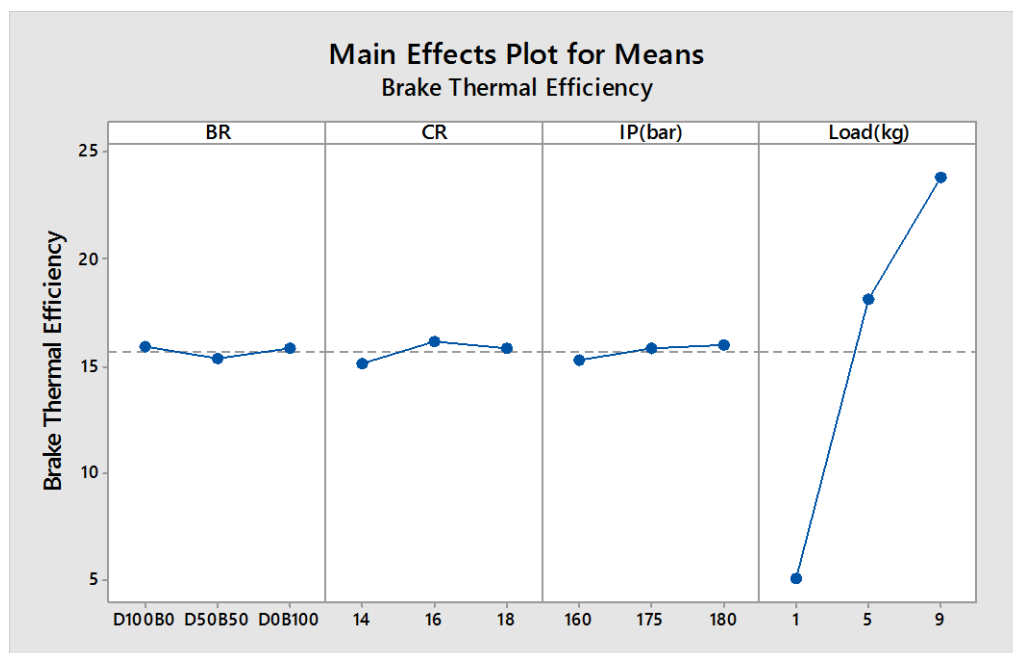


Fig 6: Main Effects Plot for Means for Brake Thermal Efficiency

Table 9 shows the Response table for Signal to Noise ratio(S/N) for Brake Thermal Efficiency.

Table 9: Response table for Signal to Noise ratio(S/N) for Brake Thermal Efficiency

Level	BR	CR	IP(bar)	Load(kg)
1	22.62	22.01	21.96	14.15
2	21.97	22.32	22.52	25.17
3	22.27	22.54	22.38	27.55
Delta	0.65	0.53	0.56	13.40
Rank	2	4	3	1

Referring figure 5, in which we have considered larger is better, we get largest values of SN ratio for blend ratio of pure diesel (D100B0) which is 22.6174. We get same for compression ratio of 18 which is 22.5401. We get same for injection pressure of 175 bar which is 22.5207. And we get maximum SN for load of 9 kg which is 27.5483. Referring figure 6, we get highest value of mean for blend ratio of pure diesel (D100B0) which is 15.93. We get highest value of mean for compression ratio of 16 which is 16.1267. We get highest value of mean for Injection pressure of 180 bar which is 15.9967 and we get highest value of mean in load for load of 9 kg which is 23.8533. And, we get the lowest value of mean for blend ratio of palm seed biodiesel (D0B100) which is 15.82. We get lowest value of mean for compression ratio of 14 which is 15.1533. We get lowest value of mean for Injection pressure of 160 bar which is 15.2667 and we get lowest value of mean in load for load of 1 kg which is 5.11. In table 9, values of Levels, BR, CR, IP, Load etc. are obtained by the same method as we obtained it for previous tables. Here, parameter level having the highest S/N ratio corresponds to the sets of parameters which indicates highest performance. Because, we have considered it for brake thermal efficiency which must be as high as possible. In above table, values of Levels, BR, CR, IP, Load etc. are obtained by the same method as we obtained it for table 3.

So, from above discussion, we obtain table 10 for optimized set.

Table 10: Optimized set of parameter for Brake Thermal Efficiency

BR	CR	IP(bar)	Load(kg)	η_{bthe} (%)	SN ratio
D100B0	18	175	9	24.3433	28.3647

From above table, we have been getting optimized set for pure diesel with compression ratio of 18, injection pressure of 175 bar, load of 9 kg and brake thermal efficiency of 24.3433 % and we get SN ratio of 28.3647. As experiment has been done on Parametric Optimization of CI Engine using Palm Seed Biodiesel, it is necessary to find the performance parameters by using palm seed biodiesel for above optimized set. In table below, optimized set for palm seed biodiesel is shown. Table 11 shows optimized set of parameters for brake thermal efficiency for palm seed biodiesel.

Table 11: Optimized set of parameters for Brake Thermal Efficiency for Palm Seed Biodiesel

BR	CR	IP(bar)	Load(kg)	η_{bthe} (%)	SN ratio
D0B100	18	175	9	24.2367	28.0218

From above table, we have been getting brake thermal efficiency of 24.2367 and SN ratio of 28.0218 for optimized set for palm seed biodiesel with compression ratio of 18, injection pressure of 175 bar, load of 9 kg. Table 12 shows validation results for brake thermal efficiency.

Table 12: Validation Results for Brake Thermal Efficiency.

Validation Results for Specific fuel Consumption	
For pure Diesel	
Predicted Value	Experimental Value
24.3433	23.43
For Palm Seed Biodiesel	
Predicted Value	Experimental Value
24.2367	23.56

VII. Conclusion

The Taguchi method is found to be an efficient technique for quantifying the effect of control parameters. We can come to conclusion that, for maximum mechanical efficiency, we can select optimized set which is using pure diesel with compression ratio of 14, injection pressure of 175 bar, load of 9 kg which gives mechanical efficiency of 47.98% and SN ratio, 33.8200. And, for same set, using palm seed biodiesel, we get mechanical efficiency of 47.3533% and SN ratio 33.7369. For maximum brake thermal efficiency, we can select optimized set which is using pure diesel with compression ratio of 18, injection pressure of 175 bar, load of 9 kg which gives brake thermal efficiency of 24.3433% and SN ratio, 28.3647. And, for same set, using palm seed biodiesel, we get brake thermal efficiency of 24.2367% and SN ratio 28.0218.

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